

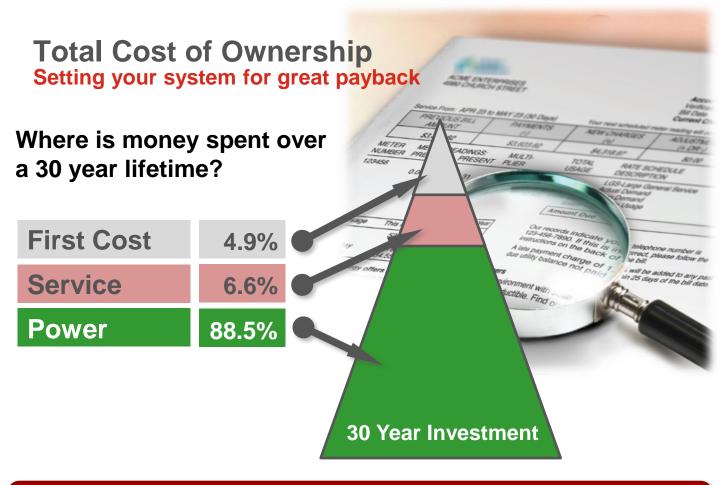
Optimized System for Your Bottom Line

(more)



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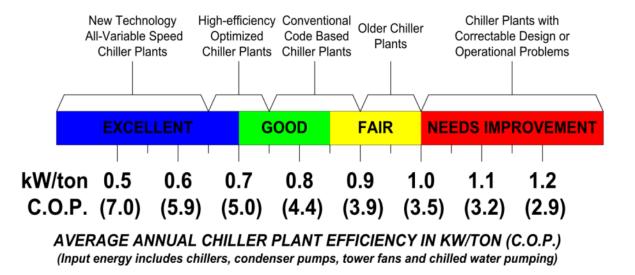




A Balanced Approach, with a Focus on Efficiency

Chiller Plant Efficiency





Note: Based on electrically driven centrifugal chiller plants in comfort conditioning application with 5.6C nominal chilled water supply temperature and open cooling towers sized for 29.4C maximum entering condenser water temperature and 20% excess capacity

Chiller Plant Efficiency





Optimal Plant Efficiency

Guaranteed chiller performance

100



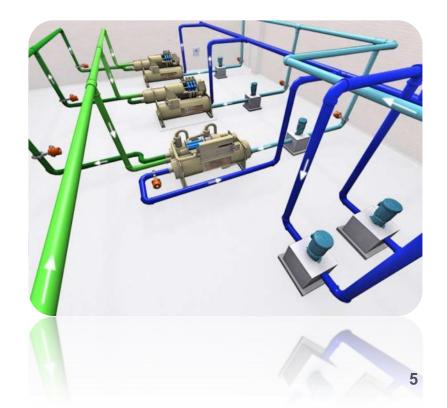
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System application & control strategy

Chiller Plant Efficiency

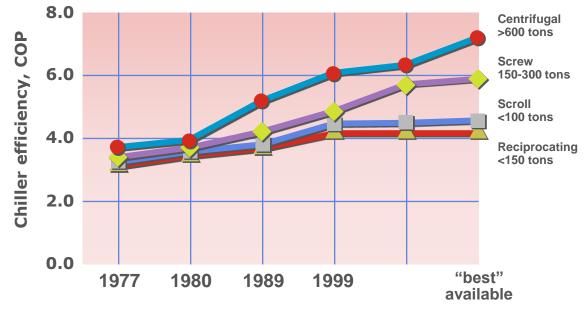


- Major Equipment for water-cooled chiller plant
 - Chiller
 - Pump
 - Cooling Tower



Chiller Performance History of Chiller Efficiency





ASHRAE Standard 90.1



Chiller Performance BEEO Requirement (2015)

Table 6.12b : Minimum Coefficient of Performance for Chiller ^{@2} at Full Load																
<u>Air-cooled</u>																
Type of compressor	Reciprocating		Scroll			Screw		VSD Screw		Centrifugal		VSD Centrifugal				
Capacity Range (kW)	Below 400 kW	400 / & ab		Below 00 kW	400 kW above		Below 500 kW & above		Below 500 kW	500 kW & above	All	All Ratings		All Ratings		
Minimum COP at cooling (free air flow ^{@1})	2.8	2.	9	2.8	2.9	2.9	2.9 3.0		2.8 (3.6) ^{@5}	2.9 (3.7) ^{@5}		3.2		3.1 (4.0) ^{@5}		
Water-cooled																
Type of compressor				Screw			VSD Screw			Centrifugal			VSD Centrifugal			
Capacity Range (kW)	Below 500 kW	500 to 1000 kW	Above 1000 kW	Below 500 kW	500 to 1000 kW	Above 1000 kW	Below 500 kW	500 t 1000 kW		Below 1000 kW	1000 kW to 3000 kW	Above 3000 kW	Below 1000 kW	1000 kW to 3000 kW	Above 3000 kW	
Minimum COP (Cooling)	4.2	4.7	5.3	4.8	5.0	5.5	4.7 (6.1) ^{@5}	4.9 (6.3)	5.2 ©5 (6.7) [©]	5.4 ^{@3} 5.6 ^{@4}	5.7	5.8	5.1 (6.6) ^{@5}	5.5 (7.1) ^{@5}	5.6 (7.2) ^{@5}	

Comprehensive review to be conducted in 2018, 2021 and 2024 respectively.

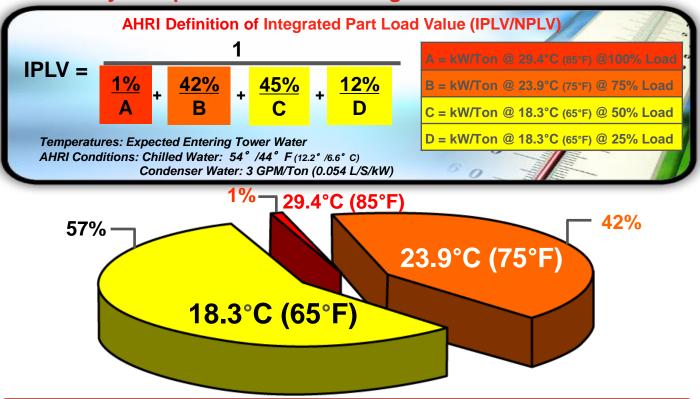


Chiller Performance Centrifugal Chiller

	Direct Drive (Hermetic Mech)	Gear Drive (Hermetic Mech)	Gear Drive (Open Mech)	Direct Drive (Hermetic Mag Lev)		
		≕- • <u></u> !.,	┉┉╇	 		
	Multi Stage	Single Stage	Single Stage	Multi Stage		
Type of Refrigerant	R-123	R-134a	R-134a	R-134a		
Theoretical Refrigerant Efficiency		0.460 kW/ton (7.6 COP)	0.460 kW/ton (7.6 COP)	0.460 kW/ton (7.6 COP)		
Centrifugal Chiller Cycle Efficiency	0.388 kW/ton (9.1 COP)	0.415 kW/ton (8.5 COP)	0.415 kW/ton (8.5 COP)	0.415 kW/ton (8.5 COP)		
Drive Train Efficiency	100%	98.1%	97.9%	100%		
Compressor Efficiency	83.3%	80.4%	81.8%	78.8%		
Motor Efficiency	95.5%	95.0%	95.0%	97.0%		
Chiller Efficiency	0.487 kW/ton (7.2 COP)	0.554 kW/ton (6.3 COP)	0.545 kW/ton (6.4 COP)	0.543 kW/ton (6.5 COP)		



Chiller Performance Efficiency Comparison – Index Rating

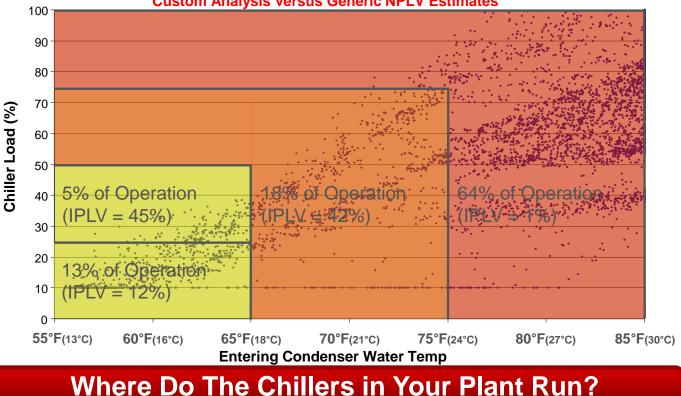


Real World Chillers Operate at Real World Conditions



Chiller Performance Efficiency Comparison – Index Rating vs. Real-World

Hong Kong (Two Chiller Plant) Custom Analysis versus Generic NPLV Estimates





Chiller Performance Real Payback Require Real Analysis



Always, Always Remember ... The Meter is On The BUILDING



Chiller Performance Real Payback Require Real Analysis



TRACE[™] 700



HVAC load design and analysis software



Compliant calculation methodologies

TPACE 700 calculations apply techniques recommended by the American Suciety of Heating, Aerlingeating and Air-Conditioning Engineers (XSHRAE). The program is tested in compliance with ASHRAE Studiated 140– 2007/2011, Standord Method of Test for the Evolution of Building Energy Analysis Compation Programs, and it areas the requirements for simulation software set by XSHRAE Standard SD.1-2007/2010 and the EUDY Coren Building Rating System. Trane Air Conditioning Economics, or TRACETM, is a design-and-analysis tool that helps HVAC professionals optimize the design of a building's heating, ventilating and air-conditioning system based on energy utilization and lifecycle cost.

A TRACE model can help establish the peak cooling and heating loads during the planning stage of a building project. At the design development stage, it aids evaluation of energy-serving concepts, such as the effects of daylighting, HVAC optimization strategies and high-performance glazing. And near the end of the construction, when the design is finalized, the TRACE model can help document compliance with ASHRAE Standard 90.1-2007/2010 or wildles the buildings rejolity for LEED* certification.

Powerful modeling capabilities

- Choose from eight load-simulation methodologies, including Heat Balance-based RIS, using algorithms provided in the latest ASHRAE Loads Toolkis. Specify either hour-by-hour (8760) or reducedyear energy/leconomic analysis.
- Choose from more than 500 predefined weather locations from around the globe.
- Describe building envelope and site orientation, as well as room construction, airflows, thermostat settings, heat sources and utilization schedules.
- Model various HVAC systems including single-zone, VAV-reheat, parallel fan-powered VAV, underfloor air distribution and dedicated outdoor-air systems.
- Model chillers, unitary equipment, water-source and geothermal heat pumps, boilers, electric resistance heating, gas-fired heat exchangers and air terminals.
- Include thermal storage, energy recovery, free cooling, cogeneration and district heating or cooling.
- Simulate control strategies, such as optimum start/stop, temperature or static pressure setpoint reset, humidification, night purge, fan cycling, demand-limiting and equipment sequencing.

Compliant calculation methodologies

TRACE 700 calculations apply techniques recommended by the American Society of Heating, Refrigerating and Air-Conditioning Engineers (ASHRAE). The program is tested in compliance with ASHRAE Standard 140-2007/2011, Standard Method of Test for the Evaluation of Building Energy Analysis

Computer Programs, and it meets the requirements for simulation software set by ASHRAE Standard 90.1-2007/2010 and the LEED® Green Building Rating System.





Chiller Plant Analyzer 12



Chiller Plant Efficiency Pump Performance

- Hydraulic Power $P_{h(kW)} = q \rho g h / (3.6 \times 10^6)$
 - $q = flow capacity (m^3/h)$
 - ρ = density of fluid (kg/m³)
 - $g = gravity (9.81 \text{ m/s}^2)$
 - h = pump head (m)
- Pump Head is the total resistance that a pump must overcome
 - Static Head
 - Friction Head
 - Pressure Head
 - Velocity Head

Pump Performance Pump Head Calculation



Never oversize pump Select pump duty point for best efficiency



jeet:	Air-to-Fresh Watar (Cooled Chiller Plant Conve	THE										
	in Peninsala Centre												
np Designati	CDWP 1 to 4							-				Date	07/19/10
ation:	R/F		_		Towrate:	118.00	,Lis	De	sign Pump Head:	26M (252 kPa)		Rev.	- 2
			Flow Rate	Nominal Pipe	Flow	Pressure Loss	Pipe Longth	Equivalent	No	No. of pipe fitting	No. of the second	Record and	Total
	Location /		Flow Hate				Pipe Length	length	Loss Factor for	No. of pape turing	venery Pressare	Pressare Loss	Accessiatio
ipe Section	Designation	Description		Size	Velocity	per Unit Length		Mahipher	Loss Pactor for Fitting				Pressare Lo
	Designation		V. (L/o	D, (mm)	v, (m/s)	pl. (Pa/m)	L (m)	Manpher	t t		Loss (kPa)	p. (kPa)	(kPa)
1-2	R.F.CDWP	Pamp Discharge	V.(0.0)	D, games	4, (88.9)	pr. (Field)	-		-	-	Loss (kraj	p. (694)	(67.8)
1-6	KT CD/HT	Pipe	118	250	2.41	160	3	-	-	-	0.48	0.48	0.48
		Batterfly Valve	118	250	2.41	160	-	-	-	-	0.58	0.58	1.06
		Check Valve	118	250	2.41	160			-		2.90	2.99	3.96
		90 deg Elbew	118	250	2.41	160	-	17	0.3		0.82	0.82	4.78
		Flexible Joint	118	250	2.41	160	-	17	0.1		0.82	0.82	5.05
2-3	RF		354	250	2.41	160	12.5			1	2.00	2.00	7.05
2.3	h r	Pipe	354	400	2.82	160		29	0.21	-	1.95	2.00	9.00
		45 deg elbew	354	400	2.82	160	-	29	0.21	2	1.95	1.95	9.00
		50 deg elbew	354				-	29					
		Tee in	354	400	2.82	160	-	29	0.3	3	4.18	4.18	14.56
		Tee out	354	400		160	-	29		-	2.37	2.37	16.93
		Reduce			2.82		-		0.3				
3-4	RF	Pipe	236	300	3.34	2.40	12.1		-	-	2.90	2.99	21.23
		45 deg elbew	236	300	3.34	240	-	22	0.2	2	2.11	2.11	23.34
		90 deg elbew	236	300	3.34	2.40	-	22	0.3	1	1.58	1.58	24.92
		Tee out	236	300	3.34	2.40	-	22	0.51	1	2.69	2.69	27.62
4.5	RF	Pipe	118	250	2.41	160	6		-	-	0.96	0.95	28.58
		Batterfly Valve	118	250	2.41	160	-		-	3	1.74	1.34	38.32
		Flexible Joint	118	250	2.41	160	-	17	0.1	2	0.54	0.54	31.86
		90 deg Elbew	118	250	2.41	160	-	17	0.3	- 4	3.26	3.26	34.12
5-6	R/F	Pipe	236	300	3.34	2.40	11		-	-	2.64	2.64	36.76
		45 deg elbew	236	300	3.34	240	-	22	0.2	2	2.11	2.11	38.88
		90 deg Elbew	236	300	3.34	240	-	22	0.3	1	1.58	1.58	48.46
		Tee in	236	300	3.34	2.40		22	0.85	1	4.49	4.49	44.95
6-7	RF	Pipe	354	400	2.82	160	12		-	-	1.92	1.92	46.87
		Reduce	354	400	2.82	160	-	29	0.2	1	0.93	0.95	47.80
		45 deg elbow	354	400	2.82	160	-	29	0.21	2	1.95	1.95	49.74
		Tee in	354	400	2.82	160	-	29	0.3	2	2.78	2.78	52.53
7-8	Cooling Tower	Pipo	118	250	2.41	160	7			-	1.12	1.12	\$3.65
		90 deg elbow	118	250	2.41	160	-	17	0.3	3	2.45	2.45	56.30
		Tee	115	250	2.41	160		17	1.11	1	3.02	3.02	59.12
		Batterfly Valve	118	250	2.41	160		17	-	1	0.58	0.58	59.70
		Pipe	59	150	3.34	2.49			-	-	2.16	2.16	61.85
		90-deg Elbew	59	150	1.14	240	-	11	0.35	2	1.85	1.85	63.70
8.9	R/F	Pipe	118	250	2.41	160	3.5			-	0.56	0.56	64.36
		Butterfly Volve	118	250	2.41	160			-	2	1.16	1.16	65.42
		90 deg elbew	118	250	2.41	160	-	17	0.3	3	2.45	2.45	67.87
9-10	R/F	Pipe	354	400	2.82	160	1			-	1.12	1.12	68.99
	Port.	90 deg Elbew	354	400	2.82	160	-	29	0.3		1.39	1.39	70.38
		Tee in	354	400	2.82	160		29	0.5	1	8.35	8.35	78.74
		Tee out	354	400	2.82	160		29	0.6	4	11.14	11.14	89.87
10-1	RF	Pipe	118	250	2.41	160	1.5			-	0.24	0.24	99.11
		Strainer	118	250	2.41	160			-		12.30	12.30	102.41
			118	250	2.41	160	-	17	0.3	-	0.58	0.58	102.41
		Batterfly Valve	118			160	-				0.58		
		90 deg Elbew		250	2.41		-	17	0.3			0.82	103.81
		Flexible Joint	118	250	2.41	160	-	17	0.1	1	0.27	0.27	104.08
		Pamp Suction											

Velocity : v (m/s) = (V / 1000) / (3.1416 * (D / 2000)²) Voocity Pressure Loss : P (kPa) = pl (Pa/m) * le (m) * x / 1000

1. The pump head calculation is according to "CIBSE Guide C" Section 4. Flow of Fluid in Pipes and Duets.
2. Values of Equivalent length of pipe 1, and Pressare loss per runt length p. are extracted from "CIBSE Guide C" Table 4.17
3. Value of Volcevity pressure loss tacter of fluing 2; in constanct flows "CIBSE Guide C" CIBSE Guide C"

Static Pressure 46.31 251.61 Total (kPa)

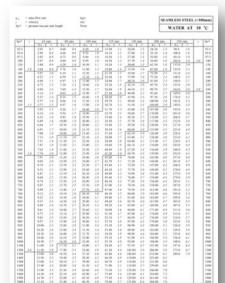




Pump Performance Pump Head Calculation – BEC 2015

BEC 2015 Chapter 6 Energy Efficiency Requirements for Air-conditioning Installation

- 6.9 Frictional Loss of Water Piping System
- 6.9.1 Water piping with diameter 50 mm or below should be sized for water flow velocity not exceeding 1.2 m/s.
- 6.9.2 Water piping with diameter larger than 50 mm should be sized for frictional loss not exceeding 400 Pa/m and
 - (a) water flow velocity not exceeding 2.5 m/s for system that operates under non-variable flow condition; or
 - (b) water flow velocity not exceeding 3.0 m/s for system that operates under variable flow condition.
 - Pressure drop per unit length
 - 2.5m/s @ 200mm pipe
 - 240 Pa/m with 89.7kg/s



Flow of Fluids in Pipes and Ducts

CIBSE Guide C4

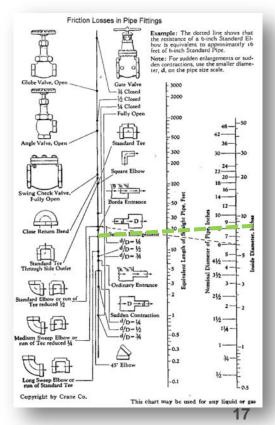




Pump Performance Pump Head Calculation

- Friction Losses in Elbow (equv. length)
 - 18 Feet (90° Elbow)
- Pressure drop
 - 18 / 3.3 x 240 Pa/m
 - = 1.309kPa
- Pump Power Consumption

 1.309 x 89.7 / 0.7 / 0.93 / 1,000
 = 0.18 kW
- Annual Operation Cost
 0.18 x 12 x 365 x 1.2
 = HKD946

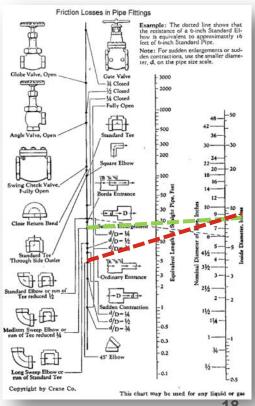




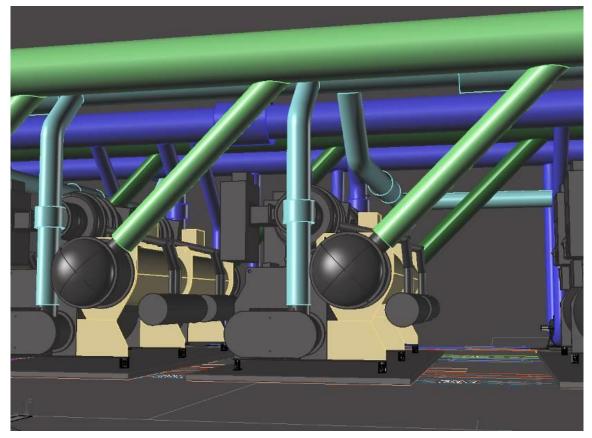
Pump Performance Pump Head Calculation

- Friction Losses in Elbow (equv. length)
 - 10 Feet (45° Elbow)
 - 18 Feet (90° Elbow)
- Pressure drop difference (18-10) / 3.3 x 240 Pa/m x 2 = 1,164Pa = 1.164kPa
- Pump Power Consumption

 1.164 x 89.7 / 0.7 / 0.93 / 1,000
 = 0.16 kW
- Annual Operation Cost
 0.16 x 12 x 365 x 1.2
 = HKD841











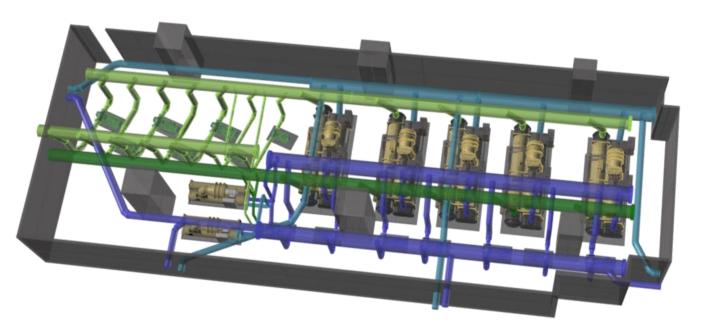












Optimal chiller plant layout and careful selection of low pressure drop devices reduces pressure losses

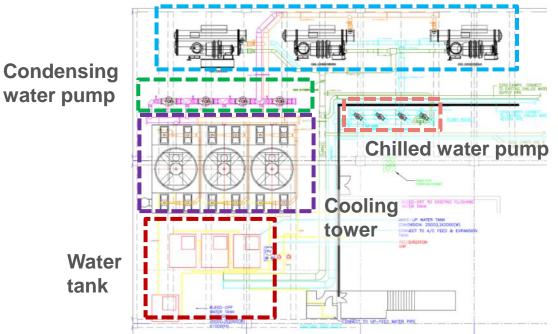


Pump Performance Simplify Piping Layout

- Friction Loss in 100% open balancing valve
 - Nominal Size: 200mm
 - Flow Rate: 89.7l/s
 - Pressure drop = 17.8kPa
- Pump Power Consumption 17.8 x 89.7 / 0.7 / 0.93 / 1,000 = 2.45 kW
- Annual Operation Cost
 2.45 x 12 x 365 x 1.2
 = HKD12,877



Pump Performance Simplify Piping Layout



Water-cooled chiller



Pump Performance Simplify Piping Layout





- Increase the pipe diameter of the system
- Minimize the length of the piping in the system
- Simplifying the layout as much as possible
- Minimize the number of elbows, tees, valves, fittings and other obstructions in the piping system
- Reduce the flow rate

Cool More or Pump More?



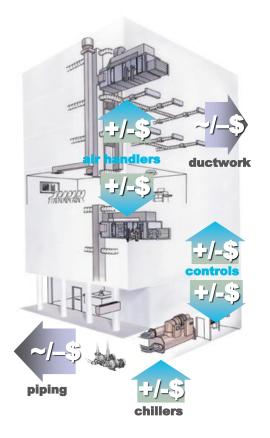
- Pump efficiency ≈ 70%
 COP ≈ 0.7
- Chiller COP ≈ 7.0
- Chiller COP ≈ 10x the pump COP

Conclusion: work your most efficient equipment harder



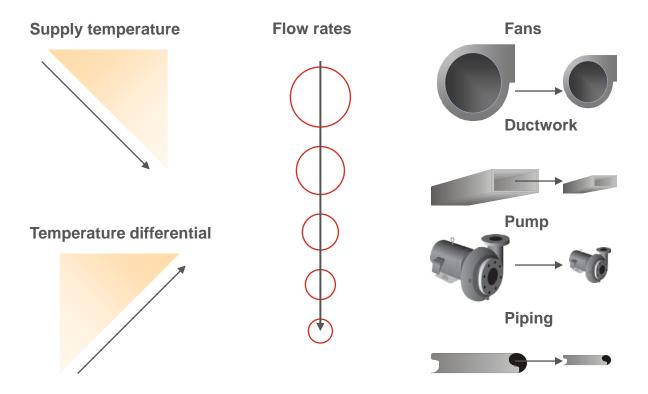
System Enhancement Earthwise Application

- Low Flow
- Low Temperature
- High Efficiency Systems



System Enhancement Earthwise Application



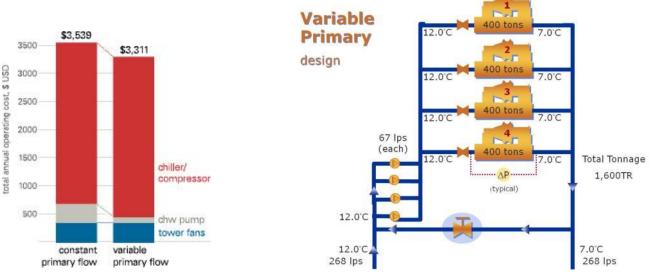




System Enhancement Why Consider Variable Primary Flow (VPF) Now?

- Chiller control sophistication
- Operating cost savings
 - Pump energy
 - Response to low ∆T Syndrome

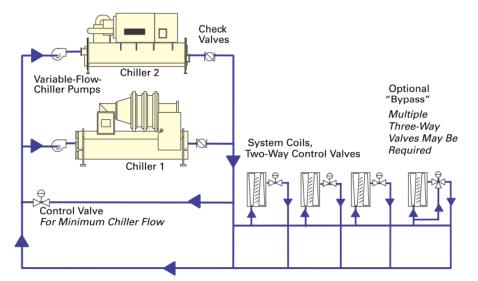




Variable Primary Flow (VPF) Advantages

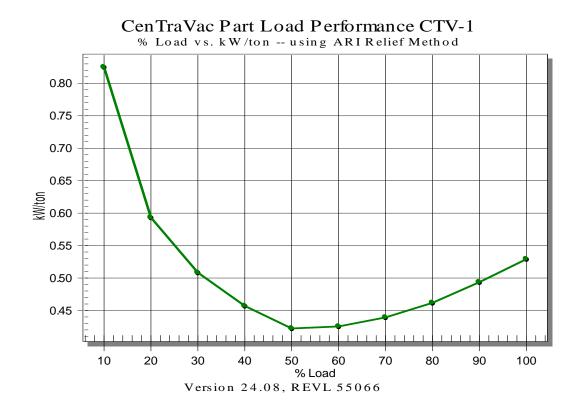


- Reduces capital investment
- Saves mechanical-room space
- Simplifies control
- Improves system reliability
- Improved chiller performance



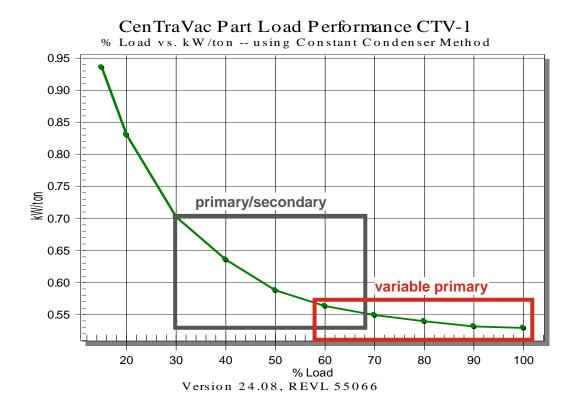


Variable Primary Flow (VPF) Improve chiller performance





Variable Primary Flow (VPF) Improve chiller performance



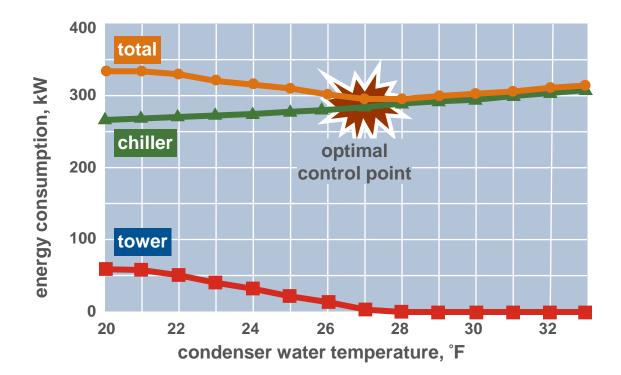
Variable Primary Flow (VPF) Three Key Application Requirements



- Chillers must be able to accommodate a change of flow of at least 10% per minute; 30% or even 50% is even better
- Minimum and maximum flows must not be violated
- A bypass is required to maintain minimum flow



Chiller-Tower Optimization (CTO) Optimal condenser water control





Chiller-Tower Optimization (CTO) Dependent On?

- Chilled water plant
 - Tower design
 - Chiller design
 - Centrifugal
 - Helical rotary
 - Variable speed drive
 - Absorption
 - Changing conditions
 - Chiller load
 - Ambient wet bulb





System Enhancement

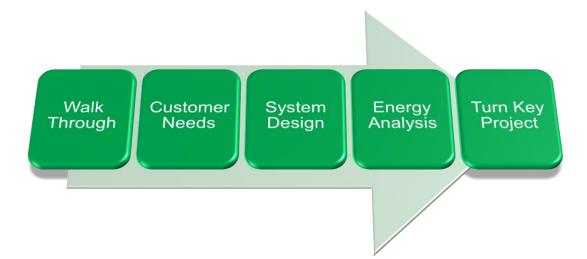


- EarthWise Application
 - Low flow, low temperature and large ΔT system
- Variable Primary Flow
- Chiller-tower Optimization



Energy Approach

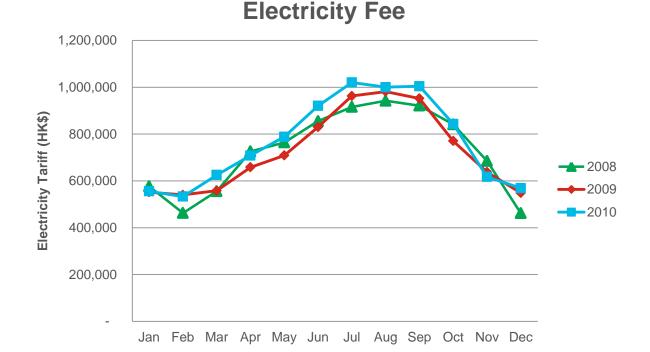




PERFORMANCE MANAGEMENT



Energy Approach Baseline Energy Consumption



Energy Approach Strategies for chiller upgrades & optimization

- Correctly Size the New Equipment
- Proper Chiller plant design
 - System Schematic
 - Layout
- Implement of Chiller Plant Control





- Determine actual building load
 - From BMS/operation log
 - Estimated from electric bill







Electricity Fee

Jan Feb Mar Apr May Jun Jul Aug Sep Oct Nov Dec

Electricity Fee for Chiller Plant = HKD5,662,191 (65% of overall)

- Determine actual building load
 - From BMS/operation log
 - Estimated from electric bill
- Downsize Chiller if possible
 - Match with cooling load profile
 - Reduce initial cost and payback period
- Replace with higher efficiency chiller
 - Improve overall savings



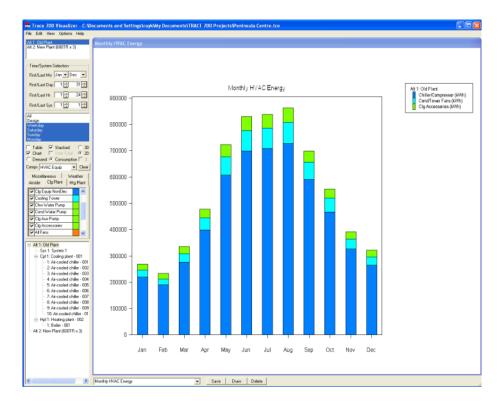




- Energy Analysis
 - Employ TRACE 700 Chiller Plant Analyzer for plant configuration comparisons
 - Input existing energy profile for analysis
 - Calculate the energy and economic effects on different configuration







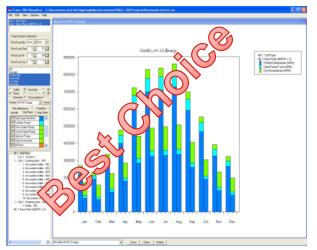


- Alternative 1
 - 2 no 600 TR water-cooled centrifugal chiller
 - 1 no 600 TR water-cooled centrifugal chiller c/w AFD
- Alternative 2
 - 2 no 750 TR water-cooled centrifugal chiller
 - 1 no 300 TR water-cooled screw chiller

Total 1,800TR cooling capacity will be provided



Alternative 1



Annual Saving: HKD 2,287,377 (40%)

HKD 2,084,079 (36.6%)

Alternative 2

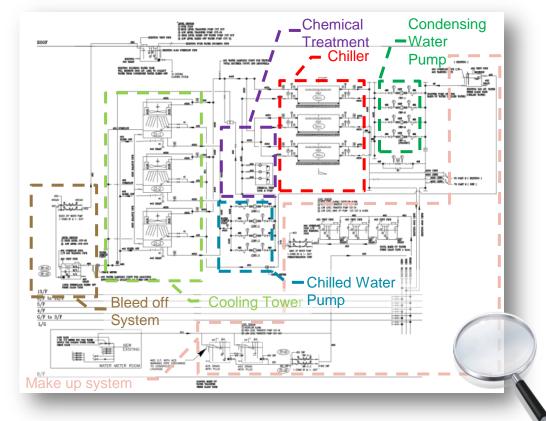
Energy Approach Chiller Plant Design (Schematic)



- Review of Existing System Arrangement
- Consideration of System Change-over / Migration
- Select chilled / condensing water distribution system
- Decide equipment design condition (e.g. Chilled Water Temp, Cooling Tower Approach)
- Properly size pipe sizes
- Associated system design (e.g. Make-up / bleed off system, chemical treatment system)



Energy Approach Chiller Plant Design (Schematic)



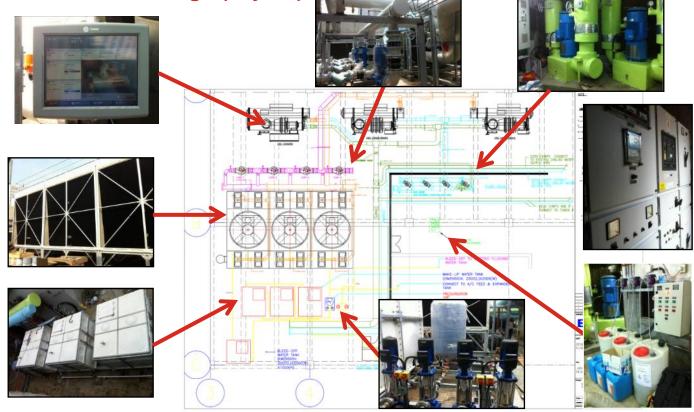


Energy Approach Chiller Plant Design (Layout)

- Satisfy statutory requirement
- Sufficient space for maintenance and service
- Minimize water pressure drop

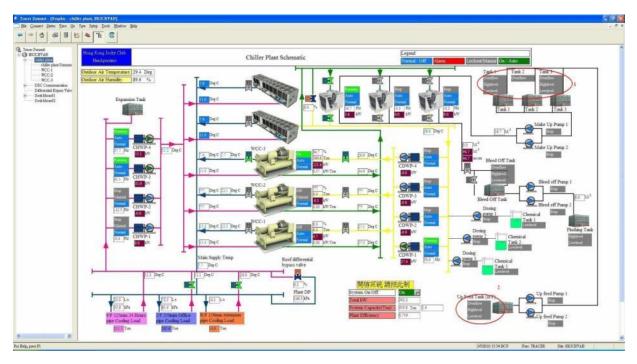


Energy Approach Chiller Plant Design (Layout)





Energy Approach Chiller Plant Control



Comprehensive control system with Cooling Tower Optimization



Energy Approach Chiller Plant Control

- Fully Automation
- Trend log for major equipment
- Alarm Management





Energy Approach Summary

- System analysis for plant configuration design
- Select high efficiency for better energy saving
- Consider pressure drop and future maintenance during pipework and layout design
- Reliable Chiller Plant Control System
- Monitor the system performance after installation



